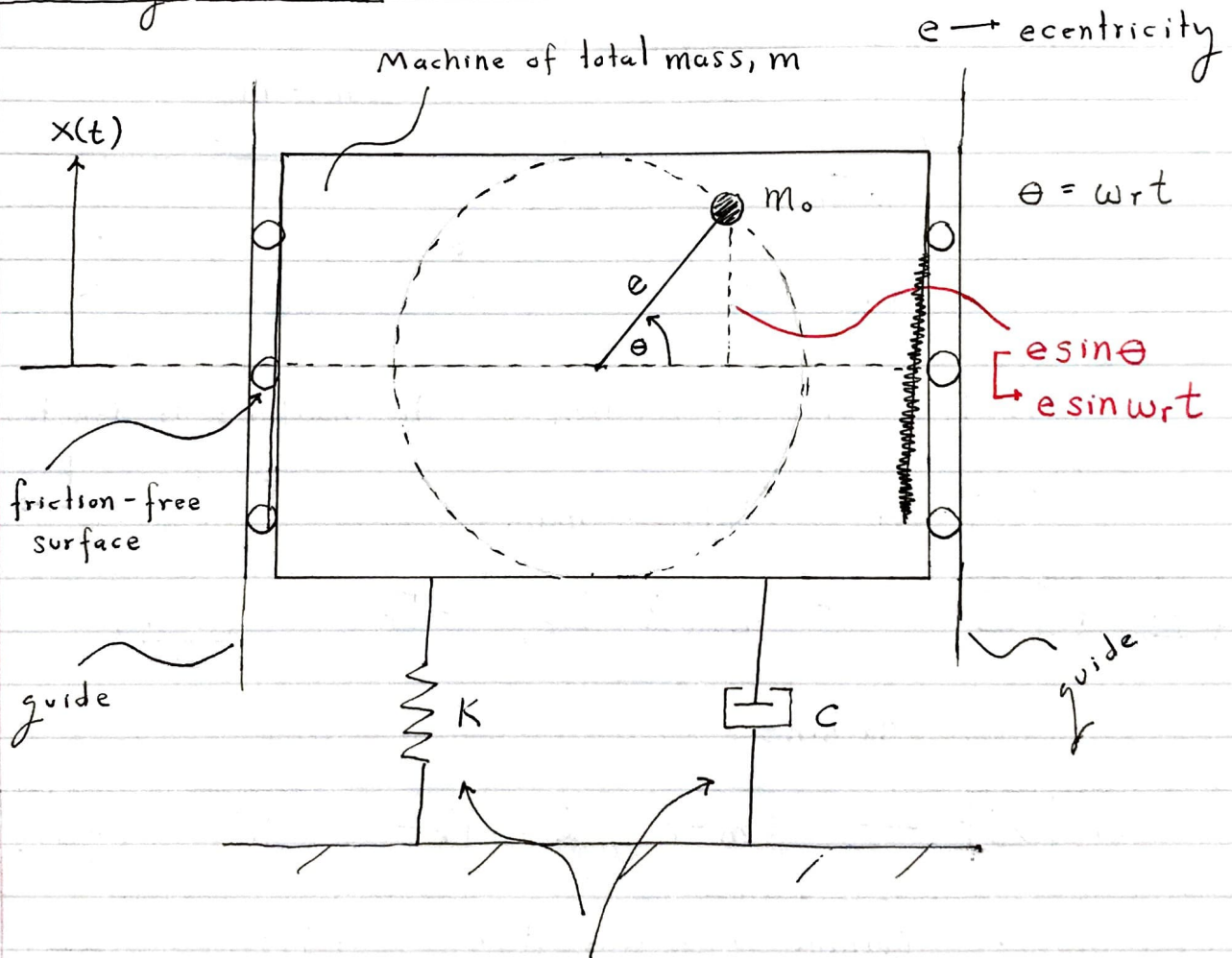
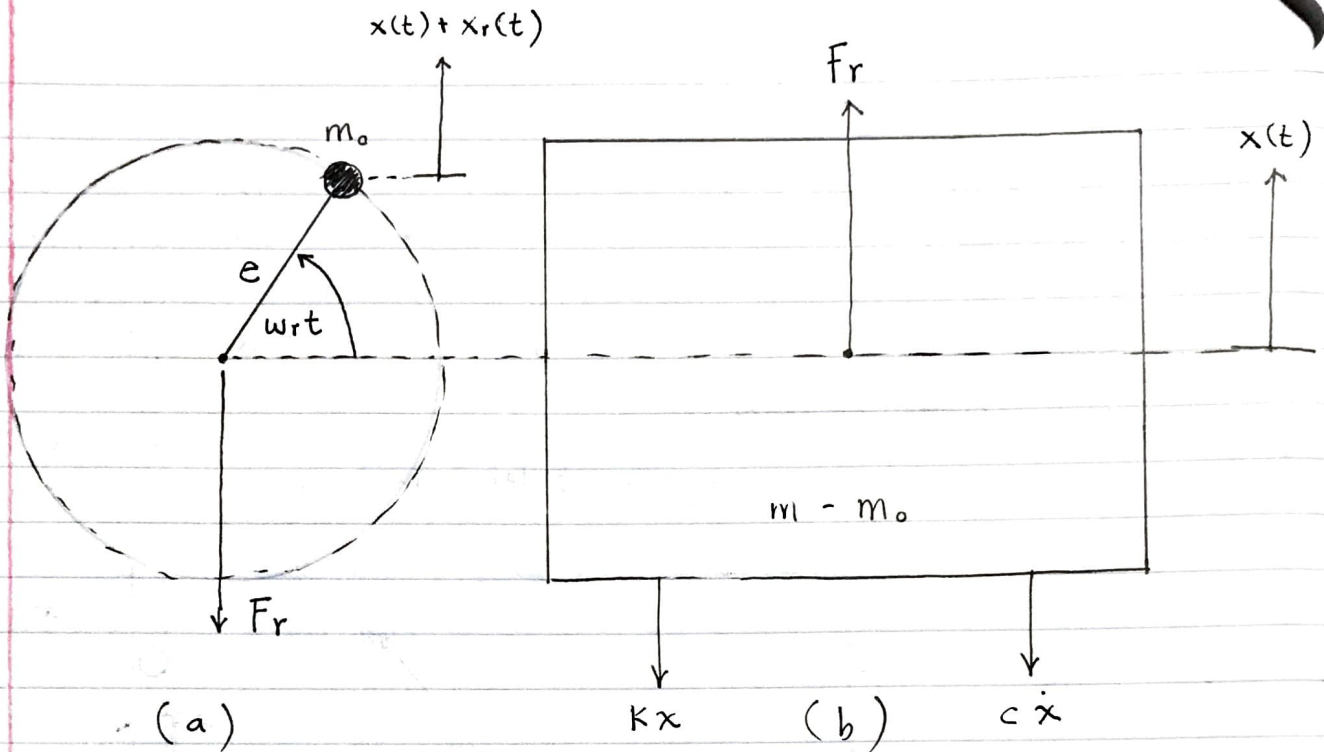


Rotating Unbalance

- Many machines & devices have rotating components, usually driven by electric motors. Small irregularities in the distribution of the mass in the rotating component can cause substantial vibration. This is called a rotating unbalance.



Rubber floor mounting modeled as a spring & a damper.



NOTE: freq. of rotation of the machine is denoted by ωr

(a) out-of-balance mass

(1) $\uparrow \sum F \rightarrow m_0(\ddot{x} + \ddot{x}_r) = -Fr$ (eq. 1)

(b) Machine

(1) $\uparrow \sum F \rightarrow (m - m_0)\ddot{x} = Fr - cx - Kx$ (eq. 2)

... combining equations 1 & 2 yields

$\rightarrow m_0(\ddot{x} + \ddot{x}_r) + (m - m_0)\ddot{x} = \cancel{Fr} + Fr - cx - Kx$

$\cancel{m_0\ddot{x}} + m_0\ddot{x}_r + m\ddot{x} - \cancel{m_0\ddot{x}} = -cx - Kx$

$m\ddot{x} + m_0\ddot{x}_r + cx + Kx = 0$ (eq. 3)

NOTE: forces in the horiz. direction are cancelled by the guides & are not considered here.

Assuming the machine rotates w/ a const. freq., ωr , the x-comp. of the motion of the mass, m_0 , is:

$$\rightarrow x_r = e \sin \omega r t \quad \therefore,$$

$$\ddot{x}_r = -e \omega r^2 \sin \omega r t \quad (\text{eq. 4})$$

... substituting into eq. 3 yields,

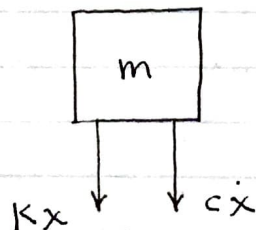
$$\boxed{m \ddot{x} + c \dot{x} + Kx = m_0 e \omega r^2 \sin \omega r t} \quad (\text{eq. 5})$$

$$F_0 = m_0 e \omega r^2$$

$$\begin{aligned} & \text{Kg} \cdot \text{m} \cdot \frac{\text{rad}^2}{\text{s}^2} \\ \hookrightarrow & = \text{Kg} \cdot \frac{\text{m}}{\text{s}^2} = \text{N} \end{aligned}$$

OR

FBD: (moving mass, m , up)



Dyn. equil. : $\Sigma F = m \ddot{x}$

$$-c\dot{x} - Kx = \cancel{m(\ddot{x} - \ddot{x}_0)} +$$

$$\frac{d^2}{dt^2} [x + e \sin \omega r t] \cdot m_0$$

NOTE: $x + e \sin \omega r t$

translation of whole machine

rotation of unbalanced mass

$$-c\dot{x} - kx = \overbrace{m(\ddot{x} - \ddot{x}_0)}^{(m-m_0)\ddot{x}} + \frac{d^2}{dt^2} [x + e \sin \omega r t] \cdot m_0$$

$$-c\dot{x} - kx = m\ddot{x} - \cancel{m_0\ddot{x}} + \cancel{m_0\ddot{x}} - m_0 e \omega r^2 \sin \omega r t$$

$$m\ddot{x} + c\dot{x} + kx = m_0 e \omega r^2 \sin \omega r t$$

F_0 (function of square)

Procedure :

NOTE: $f_0 = F_0/m$

particular solution. $\rightarrow X_p = X \sin(\omega r t - \phi)$

phase $\rightarrow \phi = \tan^{-1} \frac{2\zeta r}{1-r^2}$

magnitude $\rightarrow X = \frac{m_0 e}{m} \cdot \frac{r^2}{\sqrt{(1-r^2)^2 + (2\zeta r)^2}}$

$r \rightarrow$ freq. ratio

$$r = \frac{\omega r}{\omega_n}$$

$\zeta \rightarrow$ damping ratio

$$\zeta = \frac{c}{2\sqrt{km}}$$

$$X_p = X \sin(\omega r t - \phi) \quad (\text{eq. 6})$$

$$X = \frac{m_0 e}{m} \cdot \frac{r^2}{\sqrt{(1-r^2)^2 + (2\zeta r)^2}} \quad (\text{eq. 7})$$

$$\phi = \tan^{-1} \frac{2\zeta r}{1-r^2} \quad (\text{eq. 8})$$